



## Constructal complex-objective optimization of electromagnets based on maximization of magnetic induction and minimization of entransy dissipation rate

Lingen Chen<sup>1,2,3</sup>, Shuhuan Wei<sup>1,2,3</sup>, Zhihui Xie<sup>1,2,3</sup>, Fengrui Sun<sup>1,2,3</sup>

<sup>1</sup> Institute of Thermal Science and Power Engineering, Naval University of Engineering, Wuhan 430033, P. R. China.

<sup>2</sup> Military Key Laboratory for Naval Ship Power Engineering, Naval University of Engineering, Wuhan 430033, P. R. China.

<sup>3</sup> College of Power Engineering, Naval University of Engineering, Wuhan 430033, P. R. China.

### Abstract

An electromagnet requests high magnetic induction and low temperature. Based on constructal theory and entransy theory, a new complex-objective function of magnetic induction and mean temperature difference to describe performance of electromagnet is provided, and the electromagnet has been optimized using the new complex-objective function. When the performance of electromagnet achieves its best, the solenoid becomes longer and thinner as the number of the high thermal conductivity cooling discs increases. Simultaneously, the magnetic induction becomes higher and the mean temperature difference becomes lower. The optimized performance of electromagnet is also improved as the volume of solenoid increases. Simultaneously, as the volume of the electromagnet increases, the magnetic induction increases to its maximum and then decreases, but the mean temperature decreases all along.

*Copyright © 2015 International Energy and Environment Foundation - All rights reserved.*

**Keywords:** Constructal theory; Electromagnet; Complex-objective optimization; Entransy dissipation rate.

### 1. Introduction

Constructal theory generated at the study of configuration of flow system [1-13]. The constructal law was stated as follows: For a flow system to persist in time (to survive) it must evolve in such a way that it provides easier and easier access to the current that flow through it. The heat transfer system is an important research area for constructal theory, and the development of constructal theory proposes a new way for the research of heat conduction and convective heat transfer [14-37].

Maximum temperature is usually taken as the optimization objective in heat transfer optimization. The minimization of maximum temperature reflects the optimization result of local part (the hot spot), not the optimization result of the whole system. Some scholars used finite-time thermodynamics or entropy generation minimization (EGM) [38-43] to optimize heat transfer processes. Entropy generation minimization is a heat transfer optimization aiming at exergy lost minimization. Entropy is the measure of the conversion extent from heat to work, and entropy production is the measure of the reduction of the doing work capability due to the irreversibility of the process. The principle of minimum entropy production indicates that the stationary nonequilibrium state is characterized by the minimum entropy

production. All these concepts are discussed from the viewpoint of thermodynamics. However, what the heat conduction concerned with is the heat transport efficiency [44]. To solve this shortage in current heat transfer theory, Guo *et al.*[44] defined heat transfer potential capacity and heat transfer potential capacity dissipation function to describe the heat transfer ability amount and its dissipation rate in the heat transfer process. In terms of the analogy between heat and electrical conductions, Guo *et al.* [45] validated that heat transfer potential capacity is a new physical quantity describing heat transfer ability which is corresponding to electrical potential energy:

$$E_{vh} = \frac{1}{2} Q_{vh} U_h = \frac{1}{2} Q_{vh} T \quad (1)$$

where  $Q_{vh} = Mc_v T$  is the thermal energy or the heat stored in an object with constant volume which may be referred to as the thermal charge,  $U_h$  or  $T$  represents the thermal potential. Heat transfer analyses show that the entransy of an object in a capacitor describes its heat transfer ability, as the electrical energy in a capacitor describes its charge transfer ability. Entransy dissipation occurs during heat transfer processes, as a measurement of the heat transfer irreversibility with the dissipation related thermal resistance. Biot [46] introduced a similar concept in the 1950s in his derivation of the differential conduction equation using the variation method. Eckert *et al.*[47] summarized that Biot formulates a variational equivalent of the thermal conduction equation from the ideas of irreversible thermodynamics to define a thermal potential and a variational invariant. The thermal potential plays a role analogous to the potential energy while the variational invariant is related to the concept of dissipation function. However, Biot did not further expand on the physical meaning of the thermal potential and its application to heat transfer optimization was not found later except in approximate solutions to anisotropic conduction problems. The heat transfer ability lost in heat transfer process was called as entransy dissipation, and the entransy dissipation per unit time and per unit volume was deduced as [45]:

$$\dot{E}_{h\phi} = \dot{q} \cdot \nabla T \quad (2)$$

where  $\dot{q}$  is thermal current density vector, and  $\nabla T$  is the temperature gradient. In steady-state heat conduction,  $\dot{E}_{h\phi}$  can be calculated as the difference between the entransy input and the entransy output of the object, i.e.

$$\dot{E}_{h\phi} = \dot{E}_{h\phi, in} - \dot{E}_{h\phi, out} \quad (3)$$

The entransy dissipation rate of the whole volume in the “volume to point” conduction is

$$\dot{E}_{h\phi, V} = \int_V \dot{E}_{h\phi} dV, \quad \dot{E}_{h\phi, A} = \int_A \dot{E}_{h\phi} dS \quad (4)$$

where  $\dot{E}_{h\phi, V}$  corresponds to three-dimensional model, and  $\dot{E}_{h\phi, A}$  corresponds to two-dimensional model. The equivalent thermal resistance for multi-dimensional heat conduction problems with specified heat flux boundary condition was given as follows [45].

$$R_h = \frac{\dot{E}_{h\phi, V}}{\dot{Q}_h^2}, \quad R_h = \frac{\dot{E}_{h\phi, A}}{\dot{Q}_h^2} \quad (5)$$

where  $\dot{Q}_h$  is the whole heat flow (thermal current). The corresponding mean temperature difference was defined as:

$$\Delta \bar{T} = R_h \dot{Q}_h \quad (6)$$

The concepts of entransy and entransy dissipation were used to develop the extremum principle of entransy dissipation for heat transfer optimization: For a fixed boundary heat flux, the conduction process is optimized when the entransy dissipation is minimized (minimum temperature difference), while for a fixed boundary temperature, the conduction is optimized when the entransy dissipation is maximized (maximum heat flux). The extremum principle of entransy dissipation was used in optimization of heat conduction [48,49], heat convection[50-53], radiative heat transfer [54] and heat exchanger [55-58]. The extremum principle of entransy dissipation and its application has also been reviewed by Refs.[59-63].

Chen *et al.* [64] firstly combined the extremum principle of entransy dissipation with constructal theory, and optimized the rectangular element by taking entransy dissipation rate minimization as objective. The optimization results showed that when the thermal current density in the high conductive path is linear with the length, the optimized constructs based on entransy dissipation rate minimization are the same as those based on the maximum temperature minimization, and the mean temperature is 2/3 of the maximum temperature. When the thermal current density in the high conductive path is nonlinear with the length, the optimized constructs based on entransy dissipation rate minimization are different from those based on maximum temperature difference minimization. The constructs based on entransy dissipation rate minimization could reduce the mean temperature more effectively than the constructs based on minimization of maximum temperature.

The multidisciplinary optimization of electromagnet was discussed by Gosselin and Bejan [65], the optimal geometries of electromagnet based on maximum temperature minimization for fixed magnetic induction were deduced. Chen *et al.* [66] made a further multidisciplinary optimization of electromagnet based on entransy dissipation rate minimization. The good performance of electromagnet requests high magnetic induction and low temperature. A complex-objective function based on maximization of magnetic induction and minimization of entransy dissipation rate will be discussed in this paper.

## 2. Entransy dissipation rate versus electromagnet configuration

A cylindrical coil is taken as an example in this paper. Figure 1 shows the front and side views of the solenoid. A wire is wound in many layers around a cylindrical space of radius  $R_{in}$ . The outer radius of the coil is  $R_{out}$ , and the axial length is  $2L$ . The solenoid is considered as a continuous medium in which the electrical current density  $j$  is a constant. The electrical current density inside the wire generates a one-dimensional magnetic field on the axis of symmetry of the coil. The heat generation rate per unit volume  $q'''$  is constant at the working state.

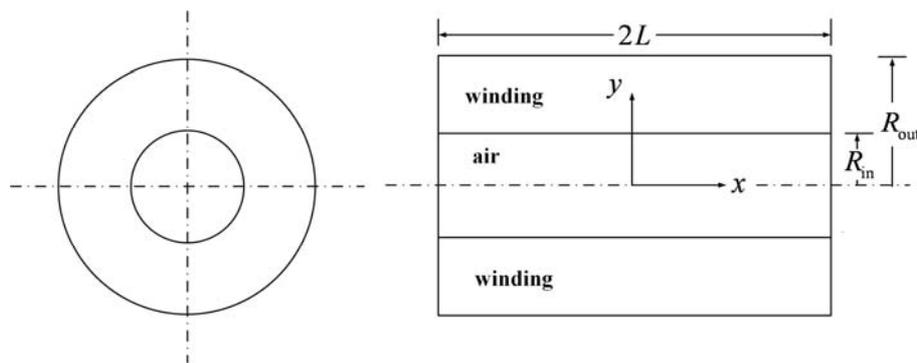


Figure 1. The main features of solenoid geometry [65]

The high thermal conductivity cooling discs of thickness  $2D$  are inserted into the solenoid to enhance heat transfer, and the discs are transversal and separate the solenoid into  $N$  sub-coils, as illustrated in Figure 2. The fraction of the volume occupied by the discs is known and fixed by

$$\phi = \frac{DN}{L} \quad (7)$$

where  $N$  is the number of discs. Most of the volume must be filled by the winding, as required by the drive toward compactness, therefore  $\phi \ll 1$ . This means that the presence of the discs does not affect significantly the magnetic field. The thermal conductivity coefficient of the material is related to its structure, density, hydrous rate, temperature, etc. But the compactness of the solenoid filled by the winding is not propitious to heat conduction; and the thermal conductivities of the wire insulating materials commonly are: polystyrene 0.08, rubber 0.202-0.29, PVC 0.17, PU 0.25, etc. The thermal conductivity of high thermal conductivity materials commonly are silver 429, copper 401, gold 401, aluminum 237, etc. The thermal conductivity of high thermal conductivity materials is defined as  $k_p$ , and the thermal conductivity of the solenoid is defined as  $k_0$ , then  $k_0 / k_p \ll 1$ . It is assumed that all the boundaries are adiabatic except the exposed external surfaces of the discs, which serve as heat sinks, the heat transfer direction in the  $k_p$  material is the  $x$ -direction, and the heat transfer direction in the  $k_0$  material is the  $r$ -direction.

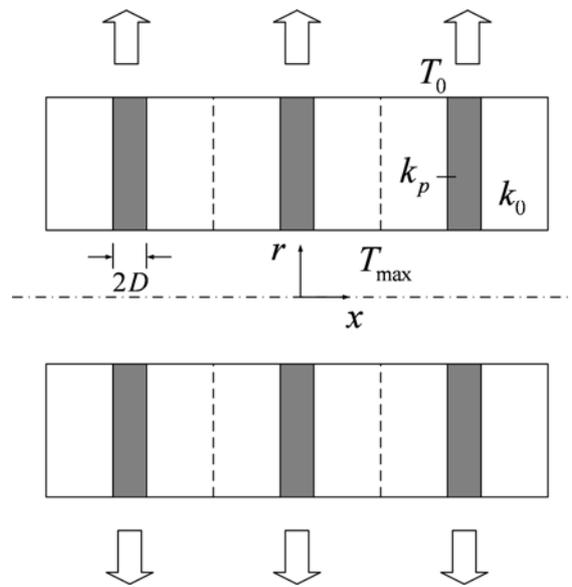


Figure 2. Solenoid cooled by transversal discs with high thermal conductivity [65]

The non-dimensional mean temperature difference based on entransy dissipation rate of solenoid is described as [66]

$$\begin{aligned}
 \Delta \tilde{T} &= \frac{\Delta \bar{T}}{P / (R_{in} k_0)} \\
 &= \frac{q'' R_{in}^2}{k_0} \left[ \frac{2 \tilde{L}^3 (\tilde{R}_{out}^2 - 1)}{3 N^2} + \frac{\tilde{L} (3 - 4 \tilde{R}_{out}^2 + \tilde{R}_{out}^4)}{4 \phi \tilde{k}} + \frac{\tilde{L} \ln \tilde{R}_{out}}{\phi \tilde{k}} \right] \cdot \frac{1}{(\tilde{R}_{out}^2 - 1) \cdot 2} \cdot \frac{R_{in} k_0}{P} \\
 &= \left[ \frac{\tilde{L} (\tilde{R}_{out}^2 - 1)}{6 N^2 \pi} + \frac{(3 - 4 \tilde{R}_{out}^2 + \tilde{R}_{out}^4)}{16 \tilde{k} \phi \pi \tilde{L}} + \frac{\ln \tilde{R}_{out}}{4 \tilde{k} \phi \pi \tilde{L}} \right] \cdot \frac{1}{(\tilde{R}_{out}^2 - 1)^2}
 \end{aligned} \tag{8}$$

The solenoid is constructural optimized based on minimization of mean temperature difference in Ref. [66]. As shown in Figure 3, the minimum mean temperature difference increases as the magnetic induction increases. The optimization results of Ref. [66] were obtained with fixed magnetic induction.

### 3. Complex-objective function of minimization of entransy dissipation rate and maximization of magnetic induction

For the case of a constant electrical current density  $j$ , the magnetic induction is given by [65]

$$\tilde{G} = 0.2 \left( \frac{2\pi\tilde{L}}{\tilde{R}_{out}^2 - 1} \right)^{1/2} \ln \frac{\tilde{R}_{out} + (\tilde{L}^2 + \tilde{R}_{out}^2)^{1/2}}{1 + (\tilde{L}^2 + 1)^{1/2}} \quad (9)$$

where

$$(\tilde{R}_{out}, \tilde{L}) = \frac{(R_{out}, L)}{R_{in}} \quad (10)$$

Eqs. (8) and (9) show that the mean temperature difference and magnetic induction are both related to electromagnet configuration. A complex-objective function to describe entransy dissipation rate and magnetic induction is defined as

$$\frac{\tilde{G}}{\Delta\tilde{T}} = \frac{0.2 \left( \frac{2\pi\tilde{L}}{\tilde{R}_{out}^2 - 1} \right)^{1/2} \ln \frac{\tilde{R}_{out} + (\tilde{L}^2 + \tilde{R}_{out}^2)^{1/2}}{1 + (\tilde{L}^2 + 1)^{1/2}}}{\left[ \frac{\tilde{L}(\tilde{R}_{out}^2 - 1)}{6N^2\pi} + \frac{(3 - 4\tilde{R}_{out}^2 + \tilde{R}_{out}^4)}{16k\phi\pi\tilde{L}} + \frac{\ln \tilde{R}_{out}}{4k\phi\pi\tilde{L}} \right] \cdot \frac{1}{(\tilde{R}_{out}^2 - 1)^2}} \quad (11)$$

The performance of electromagnet requests low entransy dissipation rate and high magnetic induction [54]. Defining  $\tilde{G}/\Delta\tilde{T}$  as the optimization objective can satisfy the request. The higher  $\tilde{G}/\Delta\tilde{T}$  is, the better the performance of electromagnet is.  $\tilde{G}/\Delta\tilde{T}$  is a complex-objective function that can describe the performance of electromagnet. It is the most important improvement of this paper comparing with those in Refs.[65, 66].

A dimensionless volume is defined as [65]

$$\tilde{V} = \frac{V}{R_{in}^3} = \pi\tilde{L}(\tilde{R}_{out}^2 - 1) \quad (12)$$

Substituting Eq. (12) into Eq. (11) yields the complex-objective function of maximization of magnetic induction and minimization of entransy dissipation rate

$$\frac{\tilde{G}}{\Delta\tilde{T}} = \frac{(2)^{1/2} \ln \left[ \tilde{R}_{out} + \left[ \left( \frac{\tilde{V}}{\pi(\tilde{R}_{out}^2 - 1)} \right)^2 + \tilde{R}_{out}^2 \right]^{1/2} \right]}{5 \left[ \frac{\tilde{V}^{1/2}}{6N^2\pi^2(\tilde{R}_{out}^2 - 1)} + \frac{(3 - 4\tilde{R}_{out}^2 + \tilde{R}_{out}^4)}{16\phi k\tilde{V}^{3/2}} + \frac{\ln \tilde{R}_{out}}{4\phi k\tilde{V}^{3/2}} \right]} \quad (13)$$

#### 4. Optimization of electromagnets

##### 4.1 Maximization of $\tilde{G}/\Delta\tilde{T}$ at different $N$

$\tilde{G}/\Delta\tilde{T}$  versus  $\tilde{R}_{out}$  at different  $N$  is shown in Figure 4. There exists a  $\tilde{R}_{out,opt}$  that  $\tilde{G}/\Delta\tilde{T}$  achieves its maximum and the performance of electromagnet achieves its best. The bigger  $N$  is, the higher  $\left( \tilde{G}/\Delta\tilde{T} \right)_{max}$  is, and the better the performance of the electromagnet is.

$\tilde{R}_{out,opt}$  and  $\tilde{L}_{opt}$  versus  $N$  when  $\tilde{G}/\Delta\tilde{T}$  achieves its maximum is shown in Figure 5.  $\tilde{R}_{out,opt}$  decreases as  $N$  increases, and  $\tilde{L}_{opt}$  increases as  $N$  increases. When the performance of electromagnet achieves its best, the solenoid becomes longer and thinner as  $N$  increases.

When the performance of electromagnet achieves its best, the corresponding  $\tilde{G}$  versus  $N$  and  $\Delta\tilde{T}$  versus  $N$  are shown in Figures 6 and 7, respectively. As  $N$  increases, the magnetic induction  $\tilde{G}$  increases and  $\Delta\tilde{T}$  decreases. The magnetic induction ability and heat transfer ability are both improved as  $N$  increases.

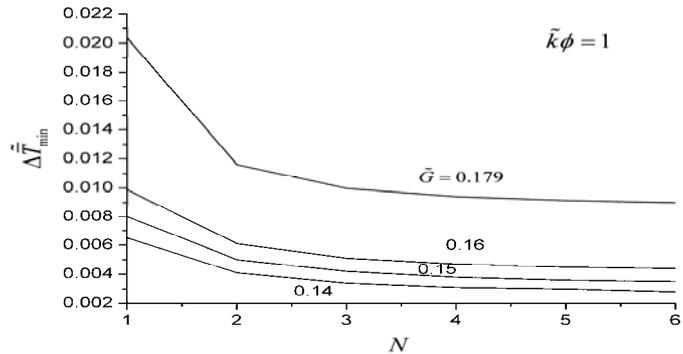


Figure 3. Effect of  $\tilde{G}$  on  $\Delta\tilde{T}_{min}$  versus  $N$  with fixed  $\tilde{k}\phi$  [66]

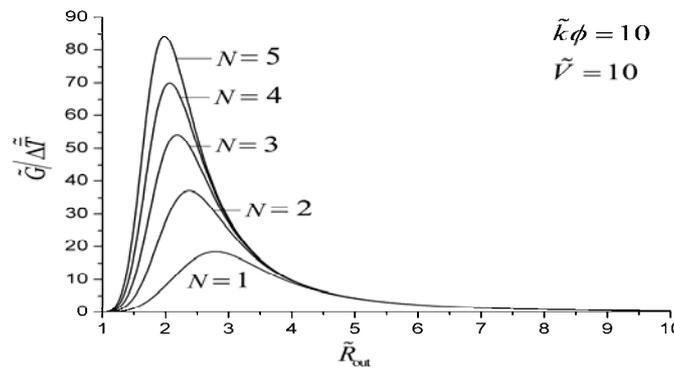


Figure 4. Effect of  $N$  on  $\tilde{G}/\Delta\tilde{T}$  versus  $\tilde{R}_{out}$  with fixed  $\tilde{k}\phi$  and  $\tilde{V}$

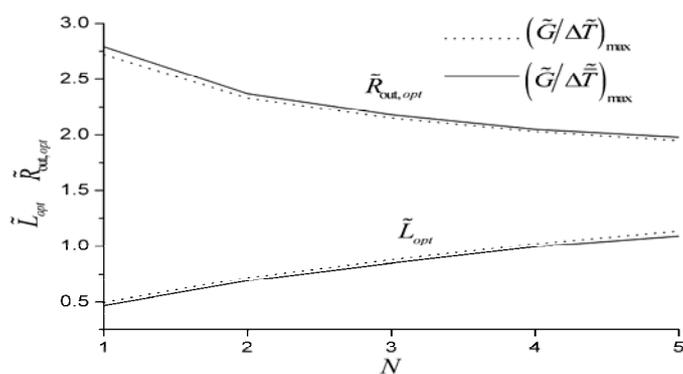


Figure 5. Comparisons between optimal geometries at  $(\tilde{G}/\Delta\tilde{T})_{max}$  and  $(\tilde{G}/\Delta\tilde{T})_{max}$

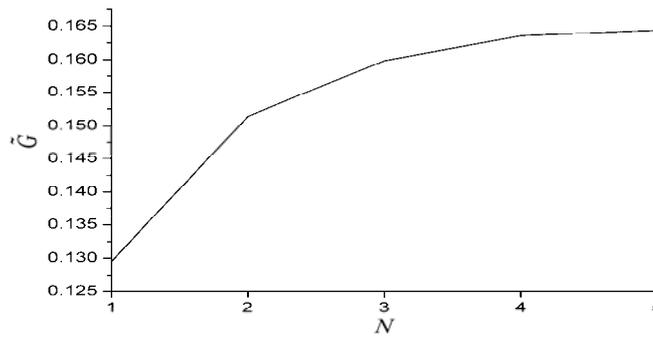


Figure 6.  $\tilde{G}$  corresponding to  $\left(\frac{\tilde{G}}{\Delta\tilde{T}}\right)_{\max}$  versus  $N$

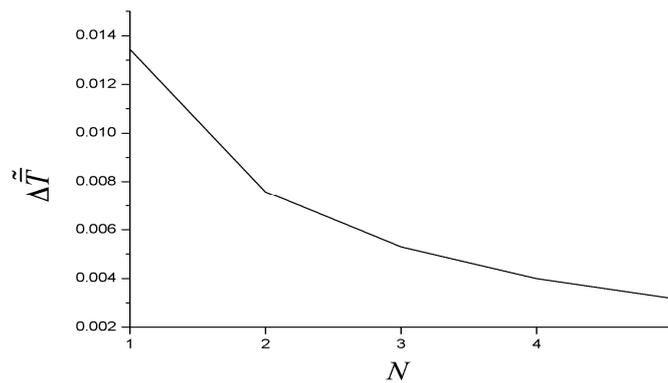


Figure 7.  $\Delta\tilde{T}$  corresponding to  $\left(\frac{\tilde{G}}{\Delta\tilde{T}}\right)_{\max}$  versus  $N$

#### 4.2 Maximization of $\tilde{G}/\Delta\tilde{T}$ at different $\tilde{V}$

$\tilde{G}/\Delta\tilde{T}$  versus  $\tilde{R}_{out}$  at different  $\tilde{V}$  is shown in Figure 8. There exists a  $\tilde{R}_{out,opt}$  that  $\tilde{G}/\Delta\tilde{T}$  achieves its maximum and the performance of electromagnet achieves its best. The bigger  $\tilde{V}$  is, the higher  $\tilde{G}/\Delta\tilde{T}$  is, and the better the performance of the electromagnet is. Figure 9 shows that the  $\left(\frac{\tilde{G}}{\Delta\tilde{T}}\right)_{\max}$  increases and approaches a constant value as  $\tilde{V}$  increases. The corresponding  $\tilde{R}_{out,opt}$  and  $\tilde{L}_{opt}$  versus  $\tilde{V}$  are shown in Figure 10. When the performance of electromagnet achieves its maximum, the corresponding  $\tilde{G}$  versus  $\tilde{V}$  and  $\Delta\tilde{T}$  versus  $\tilde{V}$  are shown in Figures 11 and 12, respectively. As  $\tilde{V}$  increases, the magnetic induction  $\tilde{G}$  increases firstly and then decreases.  $\Delta\tilde{T}$  decreases as  $\tilde{V}$  increases.  $\Delta\tilde{T}/\Delta\tilde{T}$  corresponding to different  $N$  versus  $\tilde{L}$  and  $\tilde{R}_{out}$  is shown in Figure 13, the variation of  $N, \tilde{L}$  or  $\tilde{R}_{out}$  has little impact on  $\Delta\tilde{T}/\Delta\tilde{T}$ , and  $\Delta\tilde{T}$  versus  $\Delta\tilde{T}$  keeps constant.

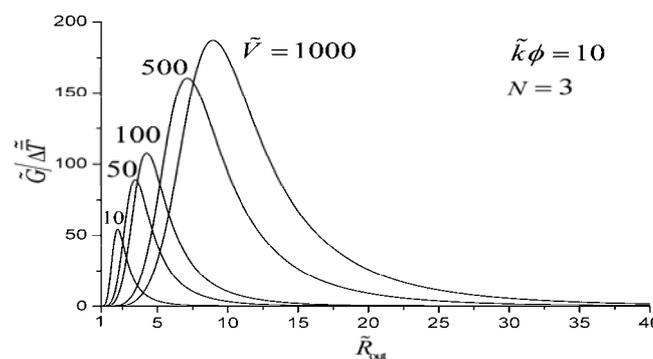


Figure 8. Effect of  $\tilde{V}$  on  $\tilde{G}/\Delta\tilde{T}$  versus  $\tilde{R}_{out}$  with fixed  $\tilde{k}\phi$  and  $N$

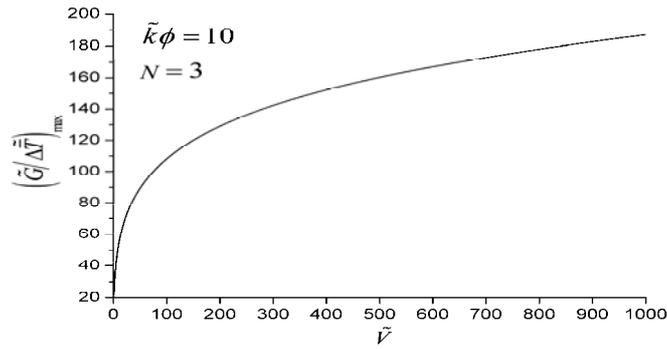


Figure 9.  $\left(\tilde{G}/\Delta\tilde{T}\right)_{\max}$  versus  $\tilde{V}$  with fixed  $\tilde{k}\phi$  and  $N$

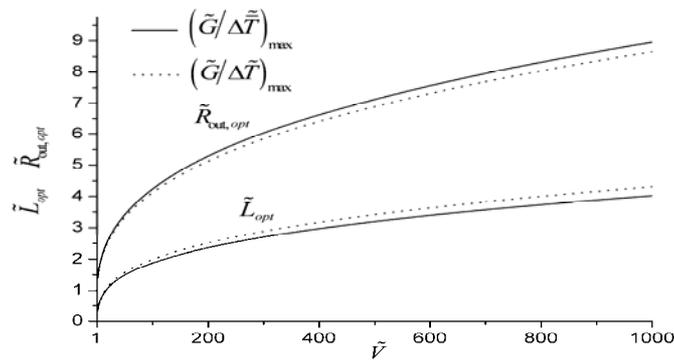


Figure 10. Comparisons between optimal geometries at  $\left(\tilde{G}/\Delta\tilde{T}\right)_{\max}$  and  $\left(\tilde{G}/\Delta\tilde{T}\right)_{\max}$

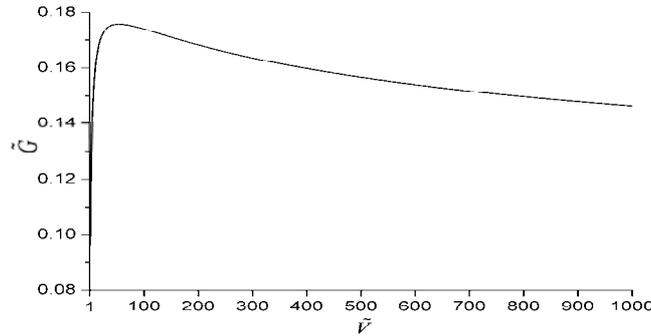


Figure 11.  $\tilde{G}$  versus  $\tilde{V}$  corresponding to  $\left(\tilde{G}/\Delta\tilde{T}\right)_{\max}$

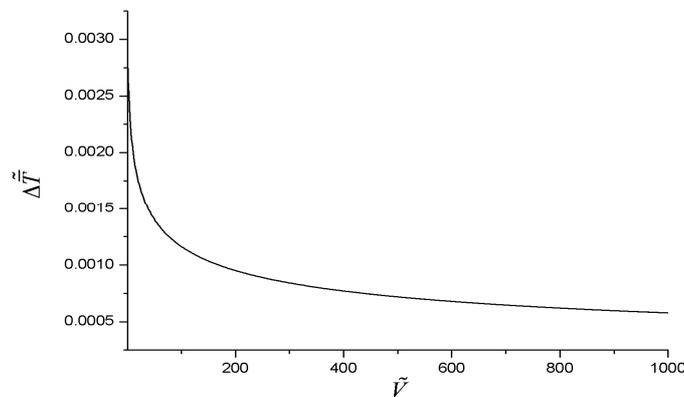


Figure 12.  $\Delta\tilde{T}$  corresponding to  $\left(\tilde{G}/\Delta\tilde{T}\right)_{\max}$  versus  $\tilde{V}$

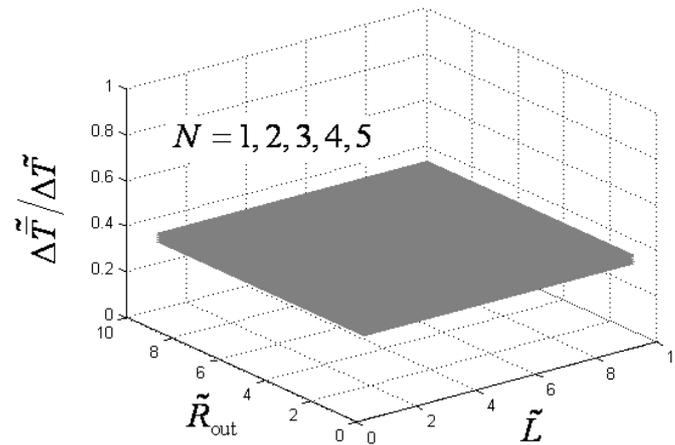


Figure 13.  $\Delta \tilde{T} / \Delta \tilde{T}$  corresponding to different  $N$  versus  $\tilde{L}$  and  $\tilde{R}_{out}$

## 5. Conclusion

Considering that the performance of electromagnet requests low entransy dissipation rate and high magnetic induction, a complex-objective function of magnetic induction and entransy dissipation rate is provided. The optimization results show that the performance of electromagnet is improved as the number  $N$  of the high thermal conductivity cooling discs inserted increases. When  $(\tilde{G} / \Delta \tilde{T})$  achieves its maximum  $(\tilde{G} / \Delta \tilde{T})_{max}$ , the solenoid becomes longer and thinner as  $N$  increases. As  $N$  increases, the magnetic induction increases and the mean temperature difference decreases.  $(\tilde{G} / \Delta \tilde{T})_{max}$  also increases as  $\tilde{V}$  increases, simultaneously, the magnetic induction increases firstly and then decreases, and the mean temperature difference decreases all along. The optimization with the complex-objective can lead to performance improvement of the electromagnet.

## Acknowledgements

This work is supported by the National Natural Science Foundation of China (Grant Nos. 51176203, 51206184 and 51356001).

## References

- [1] Bejan A. Constructal-theory network of conducting paths for cooling a heat generating volume. *Trans. ASME, J. Heat Transfer*, 1997, 40(4): 799-816.
- [2] Bejan A. *Shape and Structure, from Engineering to Nature*. Cambridge: Cambridge University Press, 2000.
- [3] Bejan A, Lorente S. Thermodynamic optimization of flow geometry in mechanical and civil engineering. *J. Non-Equilib. Thermodyn.*, 2001, 26(4): 305-354.
- [4] Rosa R N, Reis A H, Miguel A F. *Proceedings of the Symposium Bejan's Constructal Theory of Shape and Structure*. Evora: University of Evora, Portugal, 2004.
- [5] Bejan A, Lorente S. *The Constructal Law (La Loi Constructale)*, Paris: L'Harmatan, 2005.
- [6] Reis A H. Constructal theory: From engineering to physics, and how flow systems develop shape and structure. *Appl. Mech. Rev.*, 2006, 59(5): 269-282.
- [7] Bejan A, Lorente S. Constructal theory of generation of configuration in nature and engineering. *J. Appl. Phys.*, 2006, 100(4): 041301.
- [8] Bejan A, Merks G W (Editors). *Constructal Theory of Social Dynamics*. New York: Springer; 2007.
- [9] Bejan A, Lorente S. *Design with Constructal Theory*. New Jersey: Wiley, 2008.
- [10] Lorenzini G, Moretti S, Conti A. *Fin Shape Thermal Optimization Using Bejan's Constructal Theory*. San Francisco, USA: Morgan & Claypool Publishers, 2011.
- [11] Bejan A, Marden J H. The constructal unification of biological and geophysical design. *Phys. Life Rev.*, 2009, 6(2): 85-102.

- [12] Chen L. Progress in study on constructal theory and its application. *Sci. China: Tech. Sci.*, 2012, 55(3): 802-820.
- [13] Bejan A, Lorente S. Constructal law of design and evolution: Physics, biology, technology, and society. *J. Appl. Phys.*, 2013, 113(15): 151301
- [14] Gosselin L, Bejan A. Constructal heat trees at micro and nanoscales. *J. Appl. Phys.*, 2004, 96(10): 5852-5859.
- [15] Da Silva A K, Lorente S, Bejan A. Optimal distribution of discrete heat sources on a plate with laminar forced convection. *Int. J. Heat Mass Transfer*, 2004, 47(10): 2139-2148.
- [16] Wang X Q, Mujumdar A S, Yap C. Effect of bifurcation angle in tree-shaped microchannel networks. *J. Appl. Phys.*, 2007, 102(7): 073530.
- [17] Zhou S, Chen L, Sun F. Optimization of constructal economics for volume to point transport. *Appl. Energy*, 2007, 84(5): 505-511.
- [18] Lorenzini G, Moretti S. Numerical heat transfer optimization in modular systems of Y-shaped fins. *Trans. ASME, J. Heat Transfer*, 2008, 130(8): 081801.
- [19] Kim S, Lorente S, Bejan A. Dendritic vascularization for countering intense heating from the side. *Int. J. Heat Mass Transfer*, 2008, 51(25-26): 5877-5886.
- [20] Villemure C, Gosselin L, Gendron G. Minimizing hot spot temperature of porous stackings in natural convection. *Int. J. Heat Mass Transfer*, 2008, 51(15-16): 4025-4037.
- [21] Raja V A P, Basak T, Das S K. Thermal performance of a multi-block heat exchanger designed on the basis of Bejan's constructal theory. *Int. J. Heat Mass Transfer*, 2008, 51(23): 3582-3594.
- [22] Wei S, Chen L, Sun F. The volume-point constructal optimization for discrete variable cross-section conducting path. *Appl. Energy*, 2009, 86(7-8): 1111-1118.
- [23] Karakas A, Camdali U, Tunc M. Constructal optimisation of heat generating volumes. *Int. J. Exergy*, 2009, 6(5): 637-654.
- [24] Dirker J, Meyer J P. Thermal characterization of embedded heat spreading layers in rectangular heat-generating electronic modules. *Int. J. Heat Mass Transfer*, 2009, 52(5-6): 1374-1384.
- [25] Fan Y, Luo L. Second law analysis of a crossflow heat exchanger equipped with constructal distributor/collector. *Int. J. Exergy*, 2009, 6(6): 778-792.
- [26] Bello-Ochende T, Meyer J P, Bejan A. Constructal multi-scale pin-fins. *Int. J. Heat Mass Transfer*, 2010, 53(13): 2773-2779.
- [27] Marck G, Harion J L, Nemer M, et al. A new perspective of constructal networks cooling a finite-size volume generating heat. *Energy Conversion & Management*, 2011, 52(2): 1033-1046.
- [28] Hajmohammadi M R, Shirani E, Salimpour M R, Campo A. Constructal placement of unequal heat sources on a plate cooled by laminar forced convection. *Int. J. Thermal Sci.*, 2012, 60: 13-22.
- [29] Hajmohammadi M R, Poozesh S, Hosseini R. Radiation effect on constructal design analysis of a T-Y-shaped assembly of fins. *J. Thermal Sci. Tech.*, 2012, 7(4): 677-692.
- [30] Hajmohammadi M R, Poozesh S, Nourazar S S. Constructal design of multiple heat sources in a square-shaped fin. *Proc. IMechE, Part E: J. Process Mech. Eng.*, 2012, 226(4): 324-336.
- [31] Hajmohammadi M R, Poozesh S, Campo A, Nouraza S S. Valuable reconsideration in the constructal design of cavities. *Energy Conversion and Management*, 2013, 66: 33-40.
- [32] Hajmohammadi M R, Campo A, Nourazar S S, Ostad A M. Improvement of forced convection cooling due to the attachment of heat sources to a conducting thick plate, *Trans. ASME, J. Heat Transfer*, 2013, 135(12): 124504.
- [33] Hajmohammadi M R, Nourazar S S, Campo A, Poozesh S. Optimal discrete distribution of heat flux elements for in-tube laminar forced convection. *Int. J. Heat Fluid Flow*, 2013, 40: 89-96.
- [34] Hajmohammadi M R, Poozesh S, Nourazar S S, Manesh A H. Optimal architecture of heat generating pieces in a fin. *J. Mech. Sci. Techn.*, 2013, 27 (4): 1143-1149.
- [35] Hajmohammadi M R, Poozesh S, Rahmani M, Campo A. Heat transfer improvement due to the imposition of non-uniform wall heating for in-tube laminar forced convection, *Appl. Thermal Eng.*, 2013, 61(2): 268-277.
- [36] Lorenzini G, Biserni C, Rocha L A O. Constructal design of non-uniform X-shaped conductive pathways for cooling. *Int. J. Thermal Sci.*, 2013, 71: 140-147.
- [37] Hajmohammadi M R, Alizadeh Abianeh V, Moezzinajafabadi M, Daneshi M. Fork-shaped highly conductive pathways for maximum cooling in a heat generating piece. *Appl. Thermal Eng.*, 2013, 61(2): 228-235.

- [38] Andresen B. Finite-Time Thermodynamics. Physics Laboratory , University of Copenhagen, 1983.
- [39] Bejan A. Entropy Generation Minimization. New York: Wiley, 1996.
- [40] Chen L, Wu C, Sun F. Finite time thermodynamic optimization or entropy generation minimization of energy systems. *J. Non-Equilib. Thermodyn.*, 1999, 24(4): 327-359.
- [41] Berry R S, Kazakov V A, Sieniutycz S, et al. Thermodynamic Optimization of Finite Time Processes. Chichester: John Wiley & Sons Ltd., 1999.
- [42] Chen L, Sun F. Advances in Finite Time Thermodynamics: Analysis and Optimization. New York: Nova Science Publishers, 2004.
- [43] Sieniutycz S, Jezowski J. Energy Optimization in Process Systems. Oxford: Elsevier, 2009.
- [44] Guo Z, Cheng X, Xia Z. Least dissipation principle of heat transport potential capacity and its application in heat conduction optimization. *Chin. Sci. Bull.*, 2003, 48(4): 406-410.
- [45] Guo Z, Zhu H, Liang X. Entransy—A physical quantity describing heat transfer ability. *Int. J. Heat Mass Transfer*, 2007, 50(13-14): 2545-2556.
- [46] Biot M A. Variational principle in irreversible thermodynamics with applications to viscoelasticity. *Phys. Rev.*, 1955, 97(6): 1463-1469.
- [47] Eckert E R G., Drake R M. Analysis of Heat and Mass Transfer. New York: McGraw-Hill, 2004: 19-23.
- [48] Chen Q, Wang M, Pan N, Guo Z. Irreversibility of heat conduction in complex multiphase systems and its application to the effective thermal conductivity of porous media. *Int. J. Nonlinear Sci. Numer. Simul.*, 2009, 10(1): 57-66.
- [49] Xia S, Chen L, Sun F. Entransy dissipation minimization for liquid-solid phase processes. *Sci. China Ser. E: Techn. Sci.*, 2010, 53(4): 960-968.
- [50] Meng J, Liang X, Li Z. Field synergy optimization and enhanced heat transfer by multi-longitudinal vortices flow in tube. *Int. J. Heat Mass Transfer*, 2005, 48(16): 3331-3337.
- [51] Chen Q, Wang M, Pan N, Guo Z. Optimization principles for convective heat transfer. *Energy*, 2009, 34(9): 1199-1206.
- [52] Chen Q, Ren J. Generalized thermal resistance for convective heat transfer and its relation to entransy dissipation. *Chin. Sci. Bull.*, 2008, 53(23): 3753-3761.
- [53] Wang S, Chen Q, Zhang B. An equation of entransy and its application. *Chin. Sci. Bull.*, 2009, 54(19): 3572-3578.
- [54] Wu J, Liang X G. Application of entransy dissipation extremum principle in radiative heat transfer optimization. *Sci. China Ser. E: Techn. Sci.*, 2008, 51(8): 1306-1314.
- [55] Liu X, Meng, Guo Z. Entropy generation extremum and entransy dissipation extremum for heat exchanger optimization. *Chin. Sci. Bull.*, 2009, 54(6): 943-947.
- [56] Xia S, Chen L, Sun F. Optimization for entransy dissipation minimization in heat exchanger. *Chin. Sci. Bull.*, 2009, 54(19): 3587-3595.
- [57] Guo J, Cheng L, Xu M. Entransy dissipation number and its application to heat exchanger performance evaluation. *Chin. Sci. Bull.*, 2009, 54(15): 2708-2713.
- [58] Chen L, Chen Q, Li Z, Guo Z. Optimization for a heat exchanger couple based on the minimum thermal resistance principle. *Int. J. Heat Mass Transfer*, 2009, 52(21-22): 4778-4784.
- [59] Chen L. Progress in entransy theory and its applications. *Chin. Sci. Bull.*, 2012, 57(34): 4404-4426.
- [60] Chen Q, Liang X G, Guo Z Y. Entransy theory for the optimization of heat transfer – A review and update. *Inter. J. Heat Mass Transfer*, 2013, 63: 65-81.
- [61] Cheng X T, Liang X G. Entransy, entransy dissipation and entransy loss for analyses of heat transfer and heat-work conversion processes. *J. Thermal Sci. Tech.*, 2013, 8(2): 337-352.
- [62] Cheng X T, Liang X G. Entransy: its physical basis, applications and limitations. *Chin. Sci. Bull.*, 2014, 59(36): 5309-5323.
- [63] Chen L. Progress in optimization of mass transfer processes based on mass entransy dissipation extremum principle. *Sci. China: Tech. Sci.*, 2014, 57(12): 2305-2327.
- [64] Chen L, Wei S, Sun F. Constructal entransy dissipation minimization for “volume-point” heat conduction. *J. Phys. D: Appl. Phys.*, 2008, 41(19): 195506.
- [65] Gosselin L, Bejan A. Constructal thermal optimization of an electromagnet. *Int. J. Thermal Sciences*, 2004, 43(4): 331-338.

- [66] Chen L, Wei S, Sun F. Constructal entransy dissipation minimization of an electromagnet. *J. Appl. Phys.*, 2009, 105(9): 094906.



**Lingen Chen** received all his degrees (BS, 1983; MS, 1986, PhD, 1998) in power engineering and engineering thermophysics from the Naval University of Engineering, P R China. His work covers a diversity of topics in engineering thermodynamics, constructal theory, turbomachinery, reliability engineering, and technology support for propulsion plants. He had been the Director of the Department of Nuclear Energy Science and Engineering, the Superintendent of the Postgraduate School, and the President of the College of Naval Architecture and Power. Now, he is the Director, Institute of Thermal Science and Power Engineering, the Director, Military Key Laboratory for Naval Ship Power Engineering, and the Dean of the College of Power Engineering, Naval University of Engineering, P R China. Professor Chen is the author or co-author of over 1450 peer-refereed articles (over 640 in English journals) and nine books (two in English).

E-mail address: lgchenna@yahoo.com; lingenchen@hotmail.com, Fax: 0086-27-83638709 Tel: 0086-27-83615046



**Shuhuan Wei** received his BS Degrees (2003) and his PhD Degrees (2009) in power engineering and engineering thermophysics from the Naval University of Engineering, P R China. His work covers topics in engineering thermodynamics, constructal theory, reliability engineering, and technology support for propulsion plants. Dr Wei is the author or coauthor of over 30 peer-refereed articles (over 20 in English journals).



**Zhihui Xie** received his BS degree (2000) in thermal engineering and MS degree (2005) in environmental engineering from Huazhong University of Science and Technology, P R China, and received his PhD degree (2010) in power engineering and engineering thermophysics from Naval University of Engineering, P R China. His work covers topics in engineering thermodynamics and constructal theory. Associate professor Xie is the author or co-author of over 60 peer-refereed articles (over 30 in English journals).



**Fengrui Sun** received his BS Degrees in 1958 in Power Engineering from the Harbing University of Technology, P R China. His work covers a diversity of topics in engineering thermodynamics, constructal theory, reliability engineering, and marine nuclear reactor engineering. He is a Professor in the College of Power Engineering, Naval University of Engineering, P R China. Professor Sun is the author or co-author of over 850 peer-refereed papers (over 440 in English) and two books (one in English).