



Numerical simulations and analyses on thermal characteristics of 18650 lithium-ion batteries with natural cooling conditions

Fankai Meng^{1,2,3}, Linggen Chen^{1,2,3}, Zhihui Xie^{1,2,3}

¹ Institute of Thermal Science and Power Engineering, Naval University of Engineering, Wuhan 430033, P. R. China.

² Military Key Laboratory for Naval Ship Power Engineering, Naval University of Engineering, Wuhan 430033, P. R. China.

³ College of Power Engineering, Naval University of Engineering, Wuhan 430033, P. R. China.

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Abstract

As the main power source of electric vehicles and electronic devices, lithium-ion batteries have large capacity and high power density. However, the excessive heat generation often leads to high temperature of the battery, which challenges the safety of the battery. The heat generation and heat dissipation characteristics of an 18650 type lithium-ion batteries in charging process with natural cooling conditions are simulated by using finite element analysis software in this paper. The effects of charging rate and placement direction are analyzed. The results show that the maximum temperatures are 42.33 °C, 58.84 °C and 35.62 °C, i.e. increased by 7.33 °C, 23.84 °C and 1.62 °C after 1C-rate, 2C-rate and 0.5C-rate chargings, respectively. It is unable to cool the battery by natural convection cooling in 2C-rate charging. Forced cooling should be used to ensure the safety of the battery.

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Keywords: Lithium-ion batteries; Thermal characteristics; Natural convection; Simulation; Numerical heat transfer.

1. Introduction

Lithium-ion batteries are the main power sources of electric vehicles and electronic devices. They have are capacity and high power density. However, lithium-ion batteries are vulnerable to excessive heat generation during fast charging. The excessive heat generation leads to high temperature of the battery, which challenges the safety of the battery [1]. Statistics shows that most of lithium-ion battery safety accidents are caused by the overheating of the battery. Therefore, the thermal safety of lithium-ion batteries is becoming more and more important.

The temperature field of the battery under different conditions is the basis of the thermal safety evaluation of batteries. Battery performance, cycle life and system safety are all dependent on temperature distribution in the battery, which depends on heat generation rate within the battery and on heat removal rate at the battery surface. The methods of solving the temperature distribution include analytical method [2], experimental method [3], simulation method [4] and so on. The analytical method

is usually used to solve simple models while the cost of experimental method is too high. Therefore, the simulation method becomes the main method of thermal analysis.

Thermal management system is required to maintain the battery within the safe temperature range. Cooling methods for the batteries include air cooling [5-9], water cooling [10], phase change material (PCM) cooling [11], thermal pipe [12, 13], etc.. Many researchers have studied the thermal properties of lithium-ion batteries with different heat transfer conditions. Fleckenstein et al. [14] analyzed a cylindrical lithium-ion battery in detail. Jeon and Baek [15] presented transient and thermo-electric finite element analysis of cylindrical lithium-ion battery. Forgez et al. [16] developed a lumped-parameter thermal model of a cylindrical lithium-ion battery and determined heat transfer coefficients and the heat capacity. Tong et al. [10] studied the performance of a bipolar-design battery pack numerically in terms of operation and design parameters of an active thermal management system. Ye et al. [13] proposed an optimized heat pipe thermal management system for fast charging lithium-ion battery. Characteristic of mixed charge-discharge cycle is different from the charge or discharge. Lin et al. [17] investigated the electrochemical and thermal performances of a prismatic battery with a focus on the influence of temperature on cell capacity in a mixed charge–discharge cycle.

The 18650 type lithium-ion batteries are commonly used in notebook computers, or electric devices as power sources. It refers to a kind of batteries with a diameter of 18mm, length of 65mm, and cylindrical battery as shown in Figure 1 (a). Fu et al. [18] tested the 18650 lithium-ion batteries at different states of charging using a cone calorimeter to study the burning behaviors. Saw et al. [19] developed a pseudo two dimensional electrochemical coupled with lumped parameter thermal model to analyze the electrochemical and thermal behavior of the commercial 18650 lithium-ion phosphate batteries.

However, research on three-dimensional heat transfer model especially the temperature field of 18650 lithium-ion batteries with different charging rate with natural cooling conditions is rare. This paper will simulate the heat generation and heat dissipation characteristics of 18650 type lithium-ion batteries with natural cooling conditions by using a finite element analysis software ANSYS. The results may provide some guidelines for the charging management of the batteries.

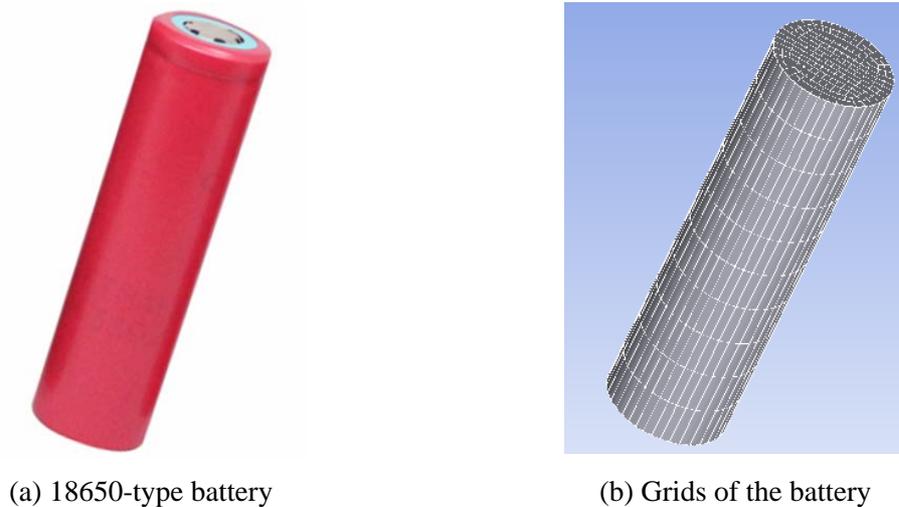


Figure 1. The 18650-type battery and the grids of the battery.

2. Physical model and parameters determination

In order to facilitate the analyses, the battery is simplified as a uniform heat generation cylinder, and satisfies the following assumptions:

- (1) Materials are uniform and isotropic. The specific heat, thermal conductivity, density and other physical parameters are not affected by temperature or the changes of state of charging (SOC);
- (2) Internal heat convection is ignored as the internal flow of electrolyte is very poor;
- (3) The electrical current and heat generation rate are uniform when the battery is charging and discharging.

The differential equation of heat conduction for the battery temperature is [20]

$$\rho C \frac{\partial t}{\partial \tau} = \frac{1}{r} \frac{\partial}{\partial r} \left(\lambda r \frac{\partial t}{\partial r} \right) + \frac{1}{r^2} \frac{\partial}{\partial \varphi} \left(\lambda \frac{\partial t}{\partial \varphi} \right) + \frac{\partial}{\partial z} \left(\lambda \frac{\partial t}{\partial z} \right) + q_v \quad (1)$$

where t is the temperature of any point in the battery, τ is time, ρ , c , and λ are battery density, specific heat capacity and thermal conductivity, respectively. q_v is battery heat generation rate. r is battery radius. Equation (1) reflects the temperature change of every point with time and space. It can be seen that the battery physical parameters ρ , C , λ and heat generation rate q_v are needed to be determined to solve the equation.

2.1 The determination of physical property parameters

The density ρ of the battery is determined by quality and volume ratio.

Different material layers arranged closely inside the battery, so the battery thermal conductivity and other properties are affected by components properties, thickness, and mass fraction. The battery is equivalent to a kind of materials, whose material specific heat capacity and thermal conductivity are determined by the components [20]. The equivalent specific heat capacity is given by

$$C = \frac{\sum m_i C_i}{m} \quad (2)$$

where m_i and C_i are the quality and specific heat capacity of composition materials, respectively. m and C are the quality and specific heat capacity of the battery. In addition, the precise equivalent heat capacity is determined by adiabatic temperature rise experiment.

Because of the different combinations of the axial and radial materials, the thermal conductivity of the battery is anisotropy. The materials along the axial direction are in parallel, so the axial equivalent thermal conductivity λ_h is given by

$$\lambda_h = \sum_i \frac{\lambda_i \delta_i}{r} \quad (3)$$

where r is battery radius, δ_i is thickness of each layer, and λ_i is the thermal conductivity of each material.

The materials along the radial direction are in series, so the radial equivalent thermal conductivity is given by

$$\lambda_r = \frac{r}{\sum_i \frac{\delta_i}{\lambda_i}} \quad (4)$$

The thickness and thermal conductivity of each layer is provided by the battery manufacturer.

2.2 The determination of the heat generation rate

Thermal battery simulation models include electro-chemical coupling model [21-25], electric-thermal coupling model [6, 26], and thermal abuse model [27]. The most commonly used electro-thermal coupling model is adopted in this paper. According to the lithium-ion battery heat generation classical model established by Bernardi et al. [28], the heat generated inside the battery is determined by

$$Q = I[(E-U) - T \frac{\partial E}{\partial T}] \quad (5)$$

where I is the electrical current through the battery, E is the open circuit voltage in equilibrium state, U is the charging voltage, T is the battery thermodynamic temperature, and $\partial E / \partial T$ is the electric potential temperature coefficient (also called as temperature entropy coefficient), which shows the relationship between the battery voltage and the temperature. The methods of determining the electric potential temperature coefficient include direct measurement method, reversible thermal equivalent method, polarized thermal deduction method, and so on [29]. The electric potential temperature coefficients for the same manufacturer of 18650 batteries are only slightly changed with SOC. According to the test data provided by the manufacturer, the average value is used in this paper.

3. Simulations and analyses of thermal characteristics at different charge rate

In order to ensure the accuracy of the simulation, non-structured hybrid grids containing hexahedrons and tetrahedrons are used. The model includes 20784 elements and 22797 nodes as shown in Figure 1 (b). The model is solved in a 0.2m cube. The ambient temperature is set as 35 °C. The battery capacity is 2.6Ah.

Due to the temperature change in the charging process, transient simulation is needed. The transient simulation time is set according to the charging rate. The charging times at 1C, 2C and 0.5C are 1800s, 3600s and 7200, respectively. Figure 2 shows the residual curve of the iterative procedure. One can see that the iterative process is converging.

Simulation results show that the maximum temperature appears in the center of the battery. The temperature fields with different rate charging process are similar qualitatively. Figure 3 shows the monitoring point temperature variations at different charge rate. It can be seen that the monitoring point temperature rises with the progress of the charging process. The battery maximum temperatures are quite different after charging with different charging rate. The thermal characteristic of the battery will be analyzed with different charging rate.

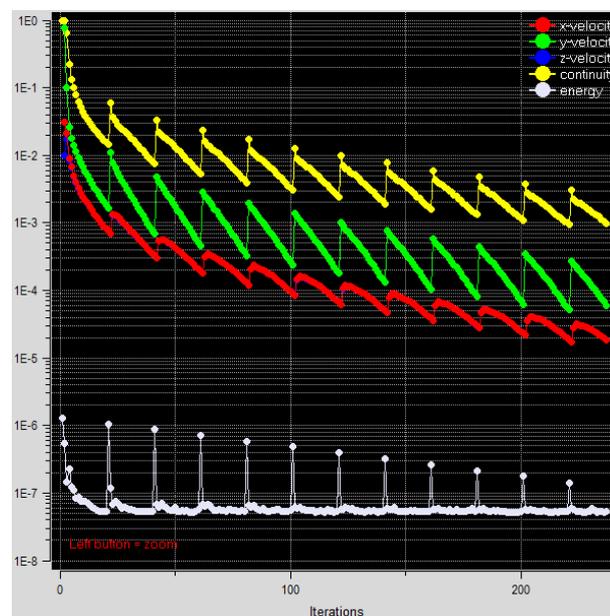


Figure 2. Residual curve of the iterative procedure.

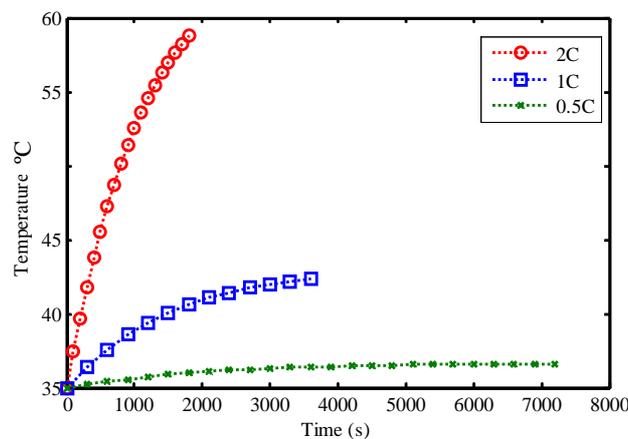


Figure 3. The battery center temperature variation during different charging rate.

3.1 1C-rate charging

Figure 4 shows the battery surface temperature contours after 1C-rate charging. It can be seen that the battery average temperature, maximum temperature and minimum temperature of each surface are

different. The maximum surface temperature is 42.32 °C appearing on the side surface while the minimum surface temperature is 42.18 °C appearing at edge of the bottom surface. Because the natural convective heat transfer of the bottom edge is the strongest, the temperature is the minimum. The natural convective heat transfer of the side surface edge is weakest, so that the temperature is the maximum.

Figure 5 shows the intermediate longitudinal section and transverse section temperature contours after 1C-rate charging. It can be seen that the internal temperature of the battery is nearly uniform. The maximum temperature is 42.33 °C, i.e. increased by 7.33 °C in the geometric center of the battery.

The temperature of air above the battery is higher while the temperatures of air at the sides and below are lower. The air temperature decreases to ambient temperature at the double diameter in the side of the battery where the heat dissipation is not affected. Because of the long charging time and small heating power, there is sufficient time for the battery internal and surface temperatures to converge. Therefore, the differences between internal temperature and the surface temperature are very small.

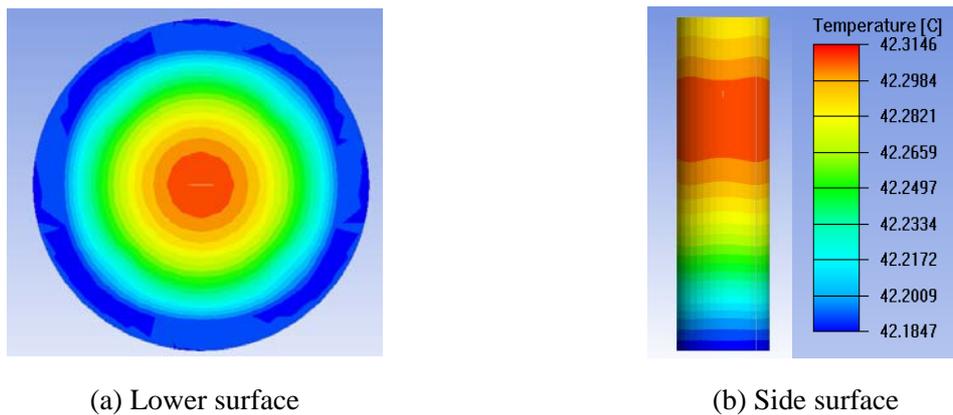


Figure 4. The surface temperature contours of the battery after 1C-rate charging.

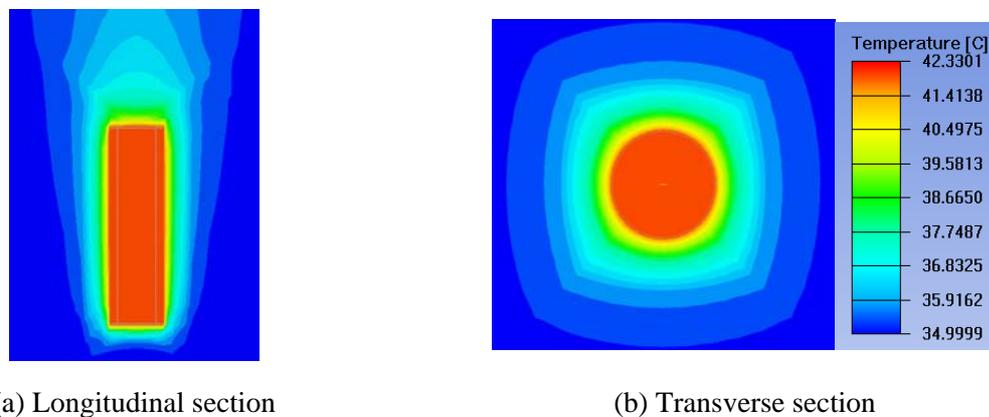


Figure 5. The section plane temperature contours of the battery after 1C-rate charging.

3.2 2C-rate charging

Figure 6 shows the intermediate longitudinal section and transverse section temperature contours after 2C-rate charging. Because of the large heat generation and short charging time, the battery temperature increases sharply in 2C-rate charging process. The maximum temperature is 58.84 °C, i.e. increased by 23.84 °C after charging. The safety temperature of the battery is 50 °C generally. It shows that it is unable to cool the battery by natural convection cooling. Therefore, forced cooling should be used to enhance the heat dissipation and ensure the safety of the battery.

3.3 0.5C-rate charging

Because the heat generation is very small and charging time is long (7200s) in 0.5C-rate charging process, the heat can be fully dissipated and the battery temperature has sufficient time to converge. Simulation shows that the battery surface and internal temperatures are both close to ambient temperature after charging. The maximum temperature is only 36.62 °C i.e. increased by 1.62 °C.

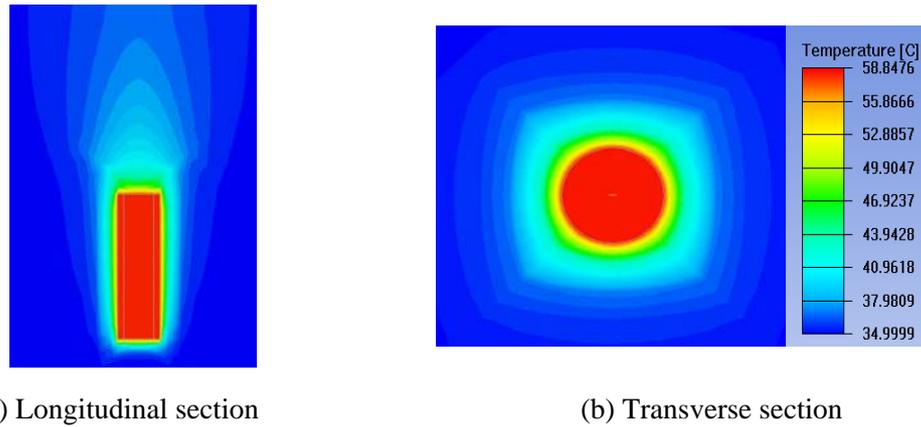


Figure 6. The section plane temperature contours of the battery after 2C-rate charging.

4. Effects of battery placement direction

The analyses mentioned above are performed when the battery is placed vertically. Figure 7 shows the surface temperature contours of the battery after 1C-rate charging when the battery is placed horizontally. The maximum surface temperature is 41.59 °C appearing on the top while the minimum surface temperature is 41.52 °C appearing at bottom edge. Figure 1 shows the intermediate transverse section and longitudinal section temperature contours when the battery is placed horizontally. It can be seen that when the battery is placed horizontally, the natural convection type changes from flat wall into rounding a cylinder, the heat transfer is enhanced. The maximum temperature is 41.61 °C, i.e. increased by 6.61 °C when charging is completed. Compared to the vertically placement, the temperature of the battery horizontally placed is decreased 0.72 °C. Thus, placement direction affects the heat transfer distinctly. Horizontal placement is more conducive than the vertical placement for heat dissipation.

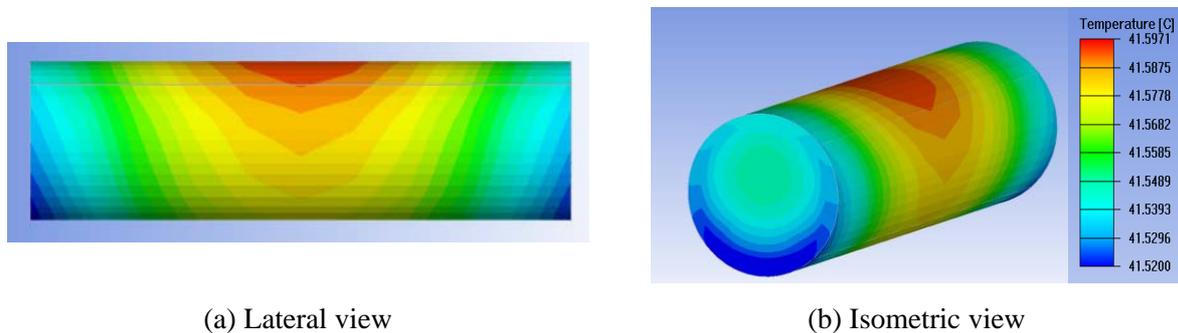


Figure 7. The surface temperature contours of the battery after 1C-rate charging when the battery is placed horizontally.

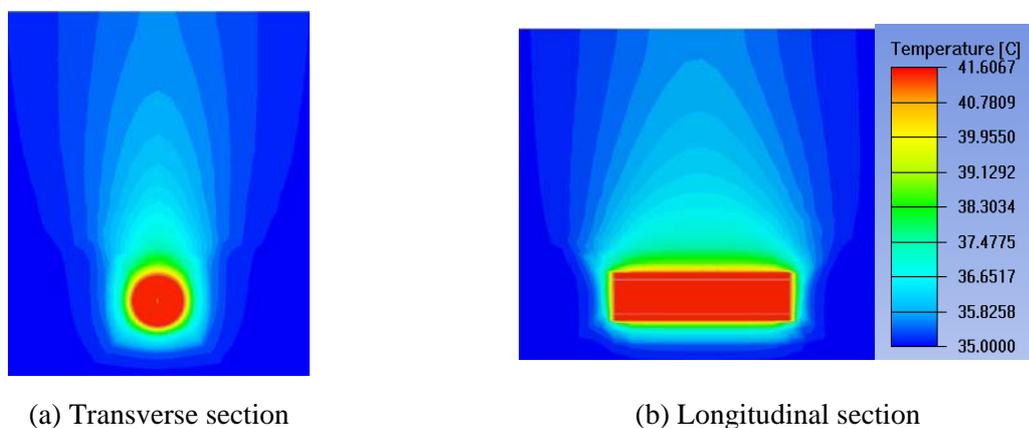


Figure 8. The intermediate transverse section and longitudinal section temperature contours when the battery is placed horizontally.

5. Conclusions

The heat generation and heat dissipation characteristics of 18650 type lithium-ion batteries in charging processes with natural cooling conditions are simulated. The effects of charging rate and placement direction are analyzed. The main conclusions are as follows:

- (1) The surface temperature and internal temperature are different after charging, but the difference is tiny.
- (2) The maximum temperature appears in the center of the battery while the minimum temperature appears at edge of the bottom surface.
- (3) The maximum temperatures are 42.33 °C, 58.84 °C and 36.62 °C, i.e. increased by 7.33 °C, 23.84 °C and 1.62 °C after 1C-rate, 2C-rate and 0.5C-rate chargings, respectively.
- (4) It is unable to cool the battery by natural convection cooling in 2C-rate charging. Forced cooling should be used to ensure the safety of the battery.

Acknowledgments

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References

- [1] Rao Z, Wang S. A review of power battery thermal energy management. *Renewable and Sustainable Energy Reviews*, 2011, 15(9): 4554-4571.
- [2] Lee C H, Bae S J, Jang M. A study on effect of lithium ion battery design variables upon features of thermal-runaway using mathematical model and simulation. *J. of Power Sources*, 2015, 293: 498-510.
- [3] Drake S J, Martin M, Wetz D A, Ostanek J K, Miller S P, Heinzl J M. Heat generation rate measurement in a Li-ion cell at large C-rates through temperature and heat flux measurements. *Journal of Power Sources*, 2015, 285: 266-273.
- [4] Inui Y, Kobayashi Y, Watanabe Y, Watase Y, Kitamura Y. Simulation of temperature distribution in cylindrical and prismatic lithium ion secondary batteries. *Energy Conversion and Management*, 2007, 48(7): 2103-2109.
- [5] Park H. A design of air flow configuration for cooling lithium ion battery in hybrid electric vehicles. *Journal of Power Sources*, 2013, 239: 30-36.
- [6] Mohammadian S K, Zhang Y. Thermal management optimization of an air-cooled Li-ion battery module using pin-fin heat sinks for hybrid electric vehicles. *J. of Power Sources*, 2015, 273: 431-439.
- [7] Yang N, Zhang X, Li G, Hua D. Assessment of the forced air-cooling performance for cylindrical lithium-ion battery packs: A comparative analysis between aligned and staggered cell arrangements. *Applied Thermal Engineering*, 2015, 80: 55-65.
- [8] Ling Z, Wang F, Fang X, Gao X, Zhang Z. A hybrid thermal management system for lithium ion batteries combining phase change materials with forced-air cooling. *Applied Energy*, 2015, 148: 403-409.
- [9] Zhao J, Rao Z, Huo Y, Liu X, Li Y. Thermal management of cylindrical power battery module for extending the life of new energy electric vehicles. *Applied Thermal Engineering*, 2015, 85: 33-43.
- [10] Tong W, Somasundaram K, Birgersson E, Mujumdar A S, Yap C. Numerical investigation of water cooling for a lithium-ion bipolar battery pack. *International Journal of Thermal Sciences*, 2015, 94: 259-269.
- [11] Babapoor A, Azizi M, Karimi G. Thermal management of a Li-ion battery using carbon fiber-PCM composites. *Applied Thermal Engineering*, 2015, 82: 281-290.
- [12] Zhao R, Gu J, Liu J. An experimental study of heat pipe thermal management system with wet cooling method for lithium ion batteries. *Journal of Power Sources*, 2015, 273: 1089-1097.
- [13] Ye Y, Saw L H, Shi Y, Tay A A O. Numerical analyses on optimizing a heat pipe thermal management system for lithium-ion batteries during fast charging. *Applied Thermal Engineering*, 2015, 86: 281-291.
- [14] Fleckenstein M, Bohlen O, Roscher M A, Ker B. Current density and state of charge inhomogeneities in Li-ion battery cells with LiFePO₄ as cathode material due to temperature gradients. *Journal of Power Sources*, 2011, 196(10): 4769 - 4778.
- [15] Jeon D H, Baek S M. Thermal modeling of cylindrical lithium ion battery during discharge cycle. *Energy Conversion and Management*, 2011, 52(8-9): 2973-2981.
- [16] Forgez C, Vinh D, Friedrich G, Morcrette M, Delacourt C. Thermal modeling of a cylindrical LiFePO₄/graphite lithium-ion battery. *Journal of Power Sources*, 2010, 195(9): 2961-2968.
- [17] Lin C, Xu S, Li Z, Li B, Chang G, Liu J. Thermal analysis of large-capacity LiFePO₄ power batteries for electric vehicles. *Journal of Power Sources*, 2015, 294: 633-642.

- [18] Fu Y, Lu S, Li K, Liu C, Cheng X, Zhang H. An experimental study on burning behaviors of 18650 lithium ion batteries using a cone calorimeter. *J. of Power Sources*, 2015, 273: 216-222.
- [19] Saw L H, Ye Y, Tay A A O. Electrochemical–thermal analysis of 18650 lithium iron phosphate cell. *Energy Conversion and Management*, 2013, 75(0): 162-174.
- [20] Incropera F, Witt D D. *Fundamentals of Heat and Mass Transfer*(6th ed.), New York: Wiley. 2007.
- [21] Kim K J, Kim J, Park M, Kwon H K, Kim H, Kim Y. Enhancement of electrochemical and thermal properties of polyethylene separators coated with polyvinylidene fluoride– hexafluoropropylene copolymer for Li-ion batteries. *J. of Power Sources*, 2012, 198: 298-302.
- [22] Amiribavandpour P, Shen W, Mu D, Kapoor A. An improved theoretical electrochemical- thermal modelling of lithium-ion battery packs in electric vehicles. *J. of Power Sources*, 2015, 284: 328-338.
- [23] Basu S, Patil R S, Ramachandran S, Hariharan K S, Kolake S M, Song T, et al. Non-isothermal electrochemical model for lithium-ion cells with composite cathodes. *Journal of Power Sources*, 2015, 283: 132-150.
- [24] Xu M, Zhang Z, Wang X, Jia L, Yang L. A pseudo three-dimensional electrochemical– thermal model of a prismatic LiFePO₄ battery during discharge process. *Energy*, 2015, 80: 303-317.
- [25] Gambhire P, Ganesan N, Basu S, Hariharan K S, Kolake S M, Song T, et al. A reduced order electrochemical thermal model for lithium ion cells. *Journal of Power Sources*, 2015, 290: 87-101.
- [26] Damay N, Forgez C, Bichat M, Friedrich G. Thermal modeling of large prismatic LiFePO₄/graphite battery. Coupled thermal and heat generation models for characterization and simulation. *Journal of Power Sources*, 2015, 283: 37-45.
- [27] Feng X, He X, Ouyang M, Lu L, Wu P, Kulp C. Thermal runaway propagation model for designing a safer battery pack with 25 Ah LiNixCoyMnzO₂ large format lithium ion battery. *Applied Energy*, 2015, 154: 74-91.
- [28] Bernardi D, Pawlikowski E, Newman J. A general energy balance for battery systems. *Journal of Electrochemical Society*, 1985, 132(1): 5-12.
- [29] Robinson J B, Darr J A, Eastwood D S, Hinds G, Lee P D, Shearing P R, et al. Non-uniform temperature distribution in Li-ion batteries during discharge-A combined thermal imaging, X-ray micro-tomography and electrochemical impedance approach. *Journal of Power Sources*, 2014, 252(0): 51-57.



Fankai Meng received all his degrees (BS, 2005; MS, 2007, PhD, 2011) in power engineering and engineering thermophysics from the Naval University of Engineering, P R China. His work covers topics in finite time thermodynamics, technology support for thermoelectric devices and thermal analysis. Dr Meng is the author or co-author of over 40 peer-refereed articles (over 20 in English journals).

E-mail address: 782601028@qq.com



Lingen Chen received all his degrees (BS, 1983; MS, 1986, PhD, 1998) in power engineering and engineering thermophysics from the Naval University of Engineering, P R China. His work covers a diversity of topics in engineering thermodynamics, constructal theory, turbomachinery, reliability engineering, and technology support for propulsion plants. He had been the Director of the Department of Nuclear Energy Science and Engineering, the Superintendent of the Postgraduate School, and the Dean of the College of Naval Architecture and Power. Now, he is the Direct, Institute of Thermal Science and Power Engineering, the Director, Military Key Laboratory for Naval Ship Power Engineering, the Direct of the National Experimental Teaching Demonstration Center for Naval Ship Power Engineering, and the Dean of the College of Power Engineering, Naval University of Engineering, P R China. Professor Chen is the author or co-author of over 1540 peer-refereed articles

(over 685 in English journals) and 12 books (two in English).

E-mail address: lgchenna@yahoo.com; lingenchen@hotmail.com, Fax: 0086-27-83638709 Tel: 0086-27-83615046



Zhihui Xie received his BS degree (2000) in thermal engineering and MS degree (2005) in environmental engineering from Huazhong University of Science and Technology, P R China, and received his PhD degree (2010) in power engineering and engineering thermophysics from Naval University of Engineering, P R China. His work covers topics in engineering thermodynamics, heat transfer and constructal theory. Associate professor Xie is the author or co-author of over 60 peer-refereed articles (over 30 in English journals).

E-mail address: zhihui-xie@163.com